

Revising the definition of anthropogenic heat flux from buildings: role of human activities and building storage heat flux

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Abstract. Buildings are a major source of anthropogenic heat emissions, impacting energy use and human health in cities. The difference in magnitude and time lag between building energy consumption and building anthropogenic heat emission is poorly quantified. Energy consumption (Q_{EC}) is a widely used proxy for the anthropogenic heat flux from buildings ($Q_{F,B}$). Here we revisit the latter's definition. If $Q_{F,B}$ is the heat emission to the outdoor environment from human activities within buildings, we can derive it from the changes in energy balance fluxes between occupied and unoccupied buildings. Our derivation shows the difference between Q_{EC} and $Q_{F,B}$ is attributable to a change in the storage heat flux induced by human activities (ΔS_{o-u_o}) (i.e., $Q_{F,B} = Q_{EC} - \Delta S_{o-u_o}$). Using building energy simulations (EnergyPlus) we calculate the energy balance fluxes for a simplified isolated building (obtaining $Q_{F,B}$, Q_{EC} , ΔS_{o-u_o}) with different occupancy states. The non-negligible differences in diurnal patterns between $Q_{F,B}$ and Q_{EC} are caused by thermal storage (e.g. hourly $Q_{F,B}$ to Q_{EC} ratios vary between -2.72 and 5.13 within a year in Beijing, China). Negative $Q_{F,B}$ can occur as human activities can reduce heat emission from a building but this is associated with a large storage heat flux. Building operations (e.g., opening windows, use of space heating and cooling system) modify the $Q_{F,B}$ by affecting not only Q_{EC} but also the ΔS_{o-u_o} diurnal profile. Air temperature and solar radiation are critical meteorological factors explaining day-to-day variability of $Q_{F,B}$. Our new approach could be used to provide data for future parameterisations of both anthropogenic heat flux and storage heat fluxes from buildings. It is evident that storage heat fluxes in cities could also be impacted by occupant behaviour.

1 Introduction

Human's activities that influence energy exchanges are critical to a wide variety of disciplines (e.g. meteorology, building design, geography, climatology, hydrology, engineering). As disciplines often have interests in different scales, purposes and/or boundary conditions, the terminology and acceptable assumptions

29 differ. However, disciplines may provide data to each other or help improve assumptions used. In this study we
30 are concerned with the interface between meteorology, climatology and building design in urban areas.
31 To model the weather and climate in urban areas, an important additional source of energy to the environment is
32 the anthropogenic heat flux (Q_F). This is defined as the heat converted from consumption of biological,
33 chemical and electrical energy and released to the atmosphere due to human activities (Oke et al., 2017). Q_F has
34 three major sources, including metabolic (people and animals) activities ($Q_{F,M}$), transport ($Q_{F,T}$) and buildings
35 ($Q_{F,B}$) (Grimmond, 1992). It can be large relative to incoming solar radiation in summer (e.g. 43% in an area of
36 Beijing (Nie et al., 2014)) and increases air temperature in cities (e.g. (Ichinose et al., 1999; Fan and Sailor,
37 2005)), subsequently contributing to higher cooling demand for buildings (Santamouris et al., 2001; Takane et
38 al., 2019). In winter Q_F can contribute to the intensity of the urban heat island (Biggart et al., 2021). Not all heat
39 generated within the building volume is directly ejected into the outdoor environment immediately but subject to
40 change in magnitude and time lag. For example, the heat generated from human activities inside buildings is
41 released initially indoors (via heating or cooling application), then transported through the building fabric by
42 conduction, allowing it to be transported into atmosphere by turbulent sensible heat flux and outgoing longwave
43 radiation. In this process the net storage heat flux (ΔQ_S) of building is modified since building fabric
44 temperature is changed by absorbing more heat from the internal heat generation.

45 In urban areas, ΔQ_S is the net uptake or release of energy from urban volume. This term is an important
46 determinant of urban climate and is regarded as a key process in the genesis of urban heat island (Goward,
47 1981). The change in building ΔQ_S is modified when heat is released by human activities but the timing of the
48 externally emissions are impacted by the building fabric characteristics and the conduction process. As prior
49 studies often use energy consumption (Q_{EC}) as a proxy for $Q_{F,B}$, derived from inventory related approaches (e.g.
50 Sailor and Lu, 2004; Iamarino et al., 2012) and building energy modelling (e.g. Heiple and Sailor, 2008; Nie et
51 al., 2014), the impact on ΔQ_S is not addressed. To qualify the ‘real’ $Q_{F,B}$ and change of ΔQ_S , we revisit the
52 definition of $Q_{F,B}$ and attempt to understand how human activities affect the energy balance fluxes of buildings.

53 If $Q_{F,B}$ is the heat released from buildings into the atmosphere as a result of human activities inside the
54 building (including human metabolism), when the building is completely unoccupied (e.g. no operational
55 appliances, no people: such as ‘ghost cities’ in China (Shepard, 2015) or vacant in Dublin (Kelly and Scott,
56 2018)); then $Q_{F,B}$ is zero. However, heat released from the unoccupied building is non-zero as there is still heat
57 exchange between building and ambient environment (see Eq. 1 and 2), as occurs in other environments with
58 large mass, such as forests (e.g. Oliphant et al., 2004), and rocks (e.g. Wang et al., 2018). $Q_{F,B}$ differs from

59 building heat emission (BHE) (e.g., Hong et al., 2020; Ferrando et al., 2021) as the latter is the total heat flux
60 released from buildings to the ambient air ($BHE_{uo} = Q_{H,uo} + Q_{BAE,uo} + L_{\downarrow,uo[air\rightarrow boi]} - L_{\uparrow,uo[boi\rightarrow air]}$) not due
61 to human activities alone. Shortwave and longwave radiation can enter the unoccupied internal building space
62 through windows and conduction through walls. It modifies the heat stored within the building volume and the
63 temperature of the building envelope and indoor air, subsequently influencing the emission of heat via sensible
64 heat flux, outgoing longwave radiation and air exchange. But this energy leaving the unoccupied building is not
65 anthropogenic heat flux. For an occupied building, the internal heat gain arises from:

- 66 (1) the equivalent sources and sinks as the unoccupied buildings; but also
67 (2) the energy linked to the indoor human activities (metabolism, powered appliances and energy inputs to
68 heating or cooling).

69 These will modify each of the energy balance fluxes. Some of this additional energy is transported out of
70 buildings through indoor-outdoor ventilation exchange and/or HVAC system, immediately contributes to $Q_{F,B}$,
71 while some is stored in the building fabric, and later is released outdoors through various pathways (convection,
72 radiation, conduction) to become $Q_{F,B}$ with a time lag. Here, we derive $Q_{F,B}$ by looking at the difference of heat
73 fluxes between occupied and unoccupied buildings.

74 If the energy balance for the building system (including the indoor air and building envelope) for an
75 unoccupied dry building (assuming latent heat is not important in this case) is:

$$76 \quad Q_{uo}^* = Q_{H,uo} + Q_{BAE,uo} + \Delta Q_{S,uo} \quad (1)$$

77 The radiation balance for an isolated unoccupied (uo) building can be expressed as:

$$78 \quad Q_{uo}^* = K_{\downarrow,uo} - K_{\uparrow,uo} + L_{\downarrow,uo} - L_{\uparrow,uo} \quad (2)$$

79 where Q^* is the net all-wave radiation, K is the shortwave radiation incoming (\downarrow) and outgoing (\uparrow) to the
80 external surfaces. The longwave (L) radiation exchanges depend on the view factors (F) between the building of
81 interest (boi), the surrounding facets of other surfaces/buildings ($other\ b$) and the sky:

$$82 \quad L_{\downarrow,uo} = L_{\downarrow,uo(F[sky\rightarrow boi])} + L_{\downarrow,uo(F[other\ b\rightarrow boi])} \quad (3)$$

$$83 \quad L_{\uparrow,uo} = L_{\uparrow,uo(F[boi\rightarrow sky])} + L_{\uparrow,uo(F[boi\rightarrow other\ b])} \quad (4)$$

84 In Eq. (1), Q_H is the turbulent sensible heat flux (convection) from external surfaces to the external ambient
85 air. Q_{BAE} is the net energy exchange from the buildings through air exchange (e.g. ventilation). When the
86 building is sealed Q_{BAE} is 0 W m^{-2} , otherwise (e.g. open windows, cracks) it can be a source or sink of energy
87 (environment \leftarrow building, or inverse). ΔQ_S is the net storage heat flux of the building volume (i.e. fabric,
88 contents, including the air). The left-hand side (LHS) of Eq. (1) is the inputs or source of energy to the building,

89 whereas the right-hand side (RHS) is the sink or energy dissipation outputs. With no human activities within the
 90 building and the internal heat generation from human and infrastructure activities is zero.

91 When the building is occupied (*o*) (e.g. appliances operating / people presence), additional terms are
 92 needed in Eq. (1) to account for the supply of energy into the building for these activities and the release of
 93 energy:

$$94 \quad Q_o^* + Q_{Internal,o} + Q_{HVAC,o} = Q_{H,o} + Q_{BAE,o} + \Delta Q_{S,o} + Q_{Waste,o} \quad (5)$$

95 The two additional sources of energy (LHS) are:

96 (1) $Q_{Internal,o}$: energy released within the building from lighting, powered appliances and metabolism (e.g.
 97 people, pets).

98 (2) $Q_{HVAC,o}$: energy consumption in the building from heating, ventilation and air conditioning (HVAC) system.

99 As the building may emit exhaust/waste heat (e.g. via HVAC systems), there is an additional sink (RHS)
 100 referred to here as $Q_{Waste,o}$. The cooling system, $Q_{Waste,o}$ will remove energy from both anthropogenic (e.g.
 101 metabolism, lighting, electrical appliance and $Q_{HVAC,o}$) and natural sources (e.g. solar radiation through
 102 windows, heat diffusion through building envelope). Thus, only the natural sources occur in both the occupied
 103 and unoccupied states. In a ‘simple’ occupied state, with HVAC operated only (i.e. no people or other
 104 appliances) there is a difference in the building storage heat flux because of the alternative route to transport this
 105 natural heat of the building out from additional source of energy.

106 Here Q_H only represents the convection heat transfer at building external surface (i.e. wall, roof and windows).

107 Both $Q_{Waste,o}$ and Q_{BAE} will be incorporated into the turbulent sensible heat flux by the time they reach the
 108 inertial sub-layer (ISL) or constant flux layer (CFL). Hence, sensors (e.g. eddy covariance or large aperture
 109 scintillometry) located in the ISL would observe this as Q_H . The separation of these three terms is to better

110 understand how human activities (e.g. open/closed windows, HVAC operation) influence each heat flux. Urban
 111 canopy parameterisation (UCP) can use this information about the separate sources and their roles in the urban
 112 energy balance to account for the modified fluxes by the time they reach the ISL. Additionally, it is clearer for
 113 multi-layer UCP where vertically the energy should enter.

114 To determine the impact of the occupancy (i.e. not just the physical building form) we can consider the
 115 difference between Eq. (5) and Eq. (1). If the radiation balance for the occupied case is:

$$116 \quad Q_o^* = K_{\downarrow,o} - K_{\uparrow,o} + L_{\downarrow,o} - L_{\uparrow,o} \quad (6)$$

117 We assume that the incoming and outgoing shortwave radiation remains unchanged because the reflectivity,
 118 transmissivity and absorptivity do not change by occupancy activities then:

119 $K_{\downarrow,o} = K_{\downarrow,u0}; \quad K_{\uparrow,o} = K_{\uparrow,u0}$

120 The incoming longwave radiation is dependent on the surroundings which are independent to the building state,

121 so:

122 $L_{\downarrow,o} = L_{\downarrow,u0}$

123 Thus, the difference in radiative fluxes between occupied and unoccupied building ($\Delta L_{\uparrow,o-u0}$) is:

124 $\Delta L_{\uparrow,o-u0} = L_{\uparrow,o} - L_{\uparrow,u0}$ (7)

125 Similarly, the difference of the heat transfer through air exchange is:

126 $\Delta BAE_{o-u0} = BAE_o - BAE_{u0}$ (8)

127 With the additional terms in Eq. (5) and the air exchanges rates difference from the activities within the
128 buildings, gives:

129 $\Delta B_{o-u0} = [Q_{Internal,o} + Q_{HVAC,o}] - [Q_{Waste,o} + \Delta BAE_{o-u0}]$ (9)

130 As the change in surface temperature influences the sensible heat fluxes and storage heat fluxes:

131 $\Delta H_{o-u0} = H_o - H_{u0}$ (10)

132 $\Delta S_{o-u0} = \Delta Q_{S,o} - \Delta Q_{S,u0}$ (11)

133 By combining the Eq. (1) and Eq. (5), we obtain:

134 $\Delta B_{o-u0} = \Delta L_{\uparrow,o-u0} + \Delta H_{o-u0} + \Delta S_{o-u0}$ (12)

135 where the LHS accounts for the net available energy as result of human activities in indoor environments and
136 the RHS shows that these impact the longwave radiation, turbulent sensible and storage heat fluxes (in this dry
137 case). With rearrangement:

138 $[Q_{Internal,o} + Q_{HVAC,o}] = \Delta S_{o-u0} + [\Delta L_{\uparrow,o-u0} + \Delta H_{o-u0} + \Delta BAE_{o-u0} + Q_{Waste,o}]$ (13)

139 The additional energy generation associated with human activities to the whole building system (LHS) is
140 apparent, as traditionally defined as $Q_{F,B}$ previously (Heiple and Sailor, 2008). Here because the heat release
141 from human metabolism indoors is considerably smaller than other sources, for simplicity of analysis, we
142 assume metabolic heat is also part of energy consumption ($Q_{EC} = Q_{Internal,o} + Q_{HVAC,o}$). Besides, some of
143 additional energy is associated with the extra gain or release of stored heat within the building volume (ΔS_{o-u0}).
144 The rest is the heat released to outdoor environment from building due to human activities, which is the $Q_{F,B}$
145 based on its definition:

146 $Q_{F,B} = \Delta L_{\uparrow,o-u0} + \Delta H_{o-u0} + \Delta BAE_{o-u0} + Q_{Waste,o}$ (14)

147 Eq. (14) demonstrates the $Q_{F,B}$ is the relative heat emission at exterior building boundary between
148 unoccupied and occupied building through longwave radiation, convection, air exchange and waste heat from

149 any mechanical heating/cooling system. The source of $Q_{F,B}$ within the building volume gives (by combining Eq.
150 (13) and Eq. (14):

$$151 \quad Q_{F,B} = Q_{EC} - \Delta S_{o-uo} \quad (15)$$

152 The sources of $Q_{F,B}$ are from both energy consumption (Q_{EC}) and difference of storage heat flux (ΔS_{o-uo})
153 between unoccupied and occupied building ($Q_{F,B}$ in this study includes part of $Q_{F,M}$ from human metabolism).
154 In most prior studies, the second term of Eq. (15) is ignored. Although the storage heat flux over a year should
155 tend to zero, over short periods (e.g. sub-daily) ΔS_{o-uo} is not zero causing time lag and magnitude difference
156 between $Q_{F,B}$ and Q_{EC} . Therefore, estimation of $Q_{F,B}$ by differences in heat emission between occupied and
157 unoccupied buildings can capture the impact of dynamic changes in the building storage heat flux especially at
158 sub-annual temporal cycle.

159 In this study, the objective is to understand the temporal profile of $Q_{F,B}$, and how and why it differs from
160 Q_{EC} at diurnal and seasonal time scales, by examining differences in energy balance fluxes between an occupied
161 and unoccupied same building. A building energy simulation tool (EnergyPlus) is used to obtain the various
162 energy balance fluxes from the building system.

163 **2 Methods**

164 **2.1 Unoccupied (uo) and occupied (o) building energy simulation (BES)**

165 Building energy simulation (BES) is widely used to estimate energy consumption, heat emission and heat
166 storage within a building, while allowing changes in heat fluxes due to human activities to be estimated. Here
167 we use EnergyPlus version 9.4 (DOE, 2020) to study an isolated building (i.e. without a surrounding
168 neighbourhood). The ASNI/ASHRAE standard 140 Case 900 test model (ASHRAE, 2017) is used, which is
169 developed in a software-to-software comparative tests for validating building thermal load. It is a 48 m² one-
170 story heavyweight rectangular prism with high mass fabrics (Appendix A), whose simple geometry is ideal to
171 understand the process of how human activities change the building energy balance fluxes in a theoretical study.
172 Modifications of the original building model for this study, include: windows are reduced to one (6 m² south-
173 facing) for more appropriate EnergyPlus single-sided ventilation calculations (Daish et al., 2016); and internal
174 heat gain, ventilation control strategy and HVAC system operation vary with different scenarios considered
175 (Table 1). For the simulations, the building is assumed to be located in Beijing as the climate has both hot
176 summer and cold winter conditions. Chinese Standard Weather Data (CSWD) selected to create a Typical

177 Meteorological Year (TMY) (China Meteorological Bureau et al., 2005) are used as the meteorological forcing,
 178 as these data are developed for simulating building thermal load and energy use.

179 The modelling scenarios (Table 1) vary with building occupation state. Two types of unoccupied (uo)
 180 buildings are considered. Neither have internal heat gains nor HVAC systems, but they differ based on air
 181 exchange between (1) unoccupied sealed (us) with no infiltration or ventilation, and (2) unoccupied ventilated
 182 (uv) with 50% of windows area kept open. The single-sided natural ventilation rate is estimated by including
 183 both wind-driven ventilation rate (V_W , $m^3 s^{-1}$) (Warren 1977):

$$184 V_W = 0.025A_{eff}U_W \quad (16)$$

185 and the stack buoyancy-driven ventilation rate (V , $m^3 s^{-1}$) (Warren 1977):

$$186 V_{Stack} = \frac{1}{3}A_{eff}C_d\sqrt{\frac{\Delta THg}{T_{ave}}} \quad (17)$$

187 where A_{eff} is the effective opening area (m^2), U_W is reference wind speed at the height of opening ($m s^{-1}$). C_d
 188 is discharge coefficient (usually taken as 0.6 (Wang and Chen, 2012)), ΔT is indoor and outdoor air temperature
 189 difference ($^{\circ}C$), H is the height of opening (m), g the gravitational acceleration ($m s^{-2}$), T_{ave} is average indoor
 190 and outdoor air temperature ($^{\circ}C$). The combined ventilation rate is (Fan et al., 2021):

$$191 V_T = \sqrt{V_W^2 + V_{Stack}^2} \quad (18)$$

192 The three occupied (o) building simulations assume occupant behaviour modifies internal heat generation,
 193 natural ventilation and HVAC systems (ov). First, $ov1$ has internal heat gains ($Q_{Internal,o}$) from human
 194 metabolism, lighting and other appliances based on local building code (MOHURD, 2018), with window always
 195 open (50%, as uv). The internal heat gains are held constant allowing the fraction of heat in $Q_{F,B}$ and ΔQ_S to be
 196 impacted by building and climate conditions but not the diurnal variability of human heat generation.

197 Second, $ov2$ considers natural ventilation based on passive cooling and thermal comfort. The window
 198 opening is controlled automatically. It is opened (50% of window area) when the indoor air temperature is
 199 higher than both outdoor air temperature and ventilation setpoint ($23^{\circ}C$ for ‘warm limit’ in bedroom
 200 (Oikonomou et al., 2012)). Otherwise, it is closed to reduce heat loss and keep the building warm. Third, since
 201 natural ventilation alone may not satisfy indoor thermal comfort, mixed mode ventilation with auxiliary HVAC
 202 system (e.g. Wang and Chen, 2013; Wang and Greenberg, 2015; Chen et al., 2017) is considered in $ov3$. The
 203 mechanical heating and cooling system are active when indoor temperature reaches the threshold ($18^{\circ}C$ for
 204 heating and $26^{\circ}C$ for cooling, MOHURD, 2018). The ventilation control strategy in $ov3$ is the same as $ov2$, but

205 the EnergyPlus hybrid ventilation manager (DOE, 2020) turns the HVAC off when natural ventilation is active
 206 to prevent simultaneous operation.

207 Table 1. Cases simulated differ based on building occupation state, internal heat gain ($Q_{Internal,o}$) and presence of natural
 208 ventilation and HVAC. Notation are defined in text and nomenclature

| Code | Occupation state | Natural ventilation | $Q_{Internal,o}$ (W m ⁻²) | Window open Temperature control (°C) | HVAC Heating/ cooling setpoint (°C) |
|------|------------------|--------------------------|--|---|--|
| us | uo | Sealed | 0 | N/A | N/A |
| uv | uo | Window always open (50%) | 0 | N/A | N/A |
| ov1 | o | Window always open (50%) | 11.8 | N/A | N/A |
| ov2 | o | Controlled ventilation | 11.8 | 23 | N/A |
| ov3 | o | Mixed mode control | 11.8 | 23 | 18/26 |

209 2.2 Determination of anthropogenic heat flux

210 The simulated hourly heat fluxes by radiation, convection, air exchange and waste heat generated from HVAC
 211 system between the isolated building and atmosphere (Table A.3) are analysed for each case (Table 2). If
 212 cooling occurs, the waste heat consists of the cooling load and electrical energy consumed by the air conditioner
 213 (Q_{HVAC}). Q_{HVAC} is predicted using a static coefficient of performance (COP) for the air conditioner, and the heat
 214 removed by an air conditioner (Q_{AC}) to the total amount of electricity consumed:

$$215 \quad Q_{HVAC,C} = \frac{Q_{AC}}{COP} \quad (19)$$

$$216 \quad Q_{Waste,C} = Q_{AC}(1 + COP^{-1}) \quad (20)$$

217 With a centralised heating system (as Beijing has), for simplicity we assume all energy associated with the
 218 heating system is released indoors, and waste heat due to boiler efficiency and pipe heat loss are not considered:

$$219 \quad Q_{HVAC,H} = Q_{HS} \quad (21)$$

$$220 \quad Q_{Waste,H} = 0 \quad (22)$$

221 Combining mechanical heating and cooling, the energy consumption and corresponding waste heat from HVAC
 222 system gives:

$$223 \quad Q_{HVAC} = Q_{HVAC,C} + Q_{HVAC,H} = \frac{Q_{AC}}{COP} + Q_{HS} \quad (23)$$

$$224 \quad Q_{Waste} = Q_{Waste,C} + Q_{Waste,H} = Q_{AC}(1 + COP^{-1}) \quad (24)$$

225 Each term in Eq. (14) is determined using an occupied (*o*) and unoccupied (*uo*) building result to determine
 226 $Q_{F,B}$ and the other fluxes. The results are analysed by season (spring (March, April and May; MAM), summer
 227 (JJA), autumn (SON) and winter (DJF)) using the median (50%) and interquartile range (IQR) between the 25th
 228 and 75th percentiles to assess the diurnal patterns.

229 2.3 Ratio of anthropogenic heat flux to energy consumption

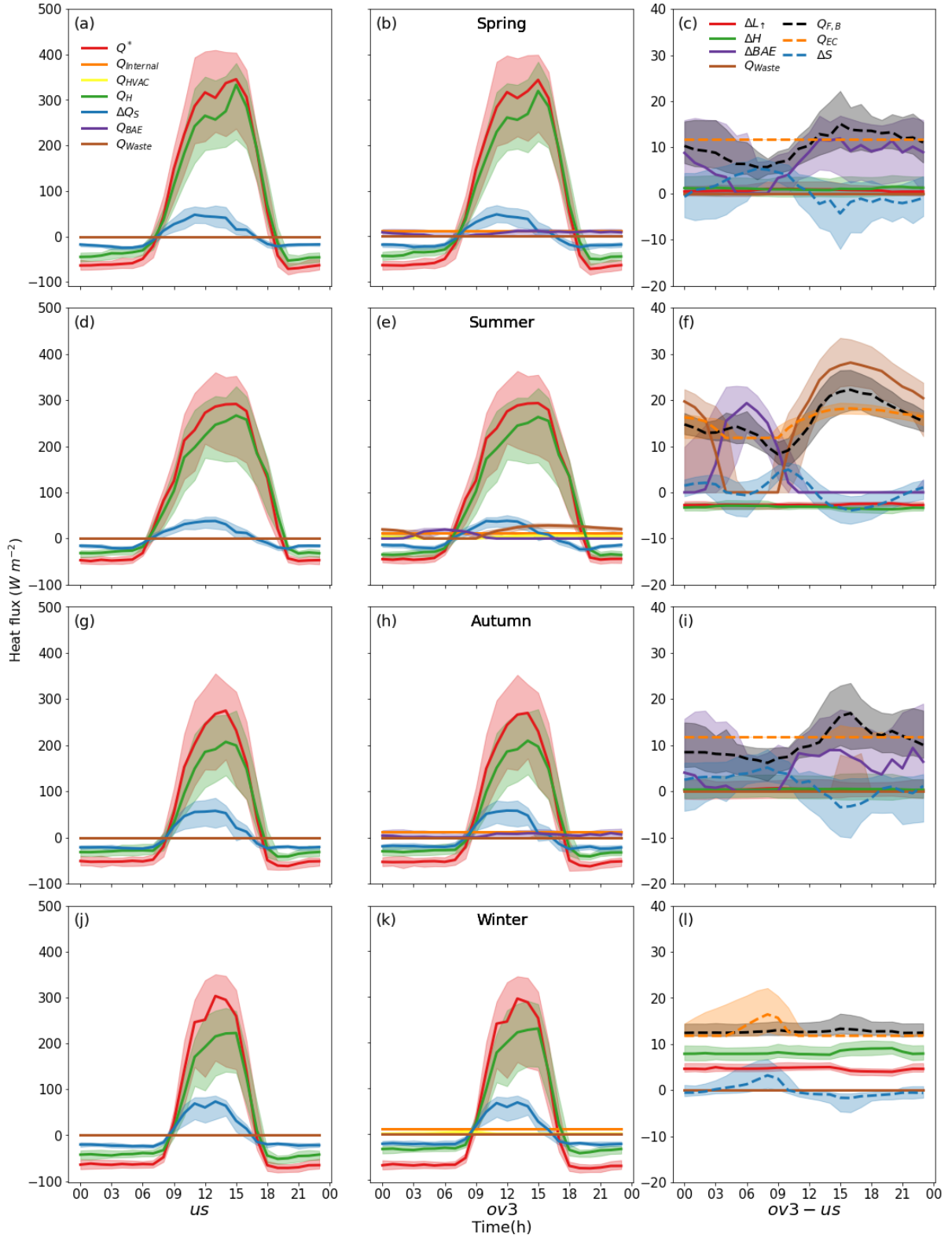
230 If the energy consumed within the building is rejected immediately into the atmosphere (Heiple and Sailor,
231 2008), the change in ΔQ_S is not accounted for, and therefore $Q_{F,B}$ is assumed to be only from energy
232 consumption (Q_{EC}). The variation of ΔQ_S associated with human activities is considered when using the relative
233 heat emissions in Eq. (14) and Eq. (15). We use the ratio $R = \frac{Q_{F,B}}{Q_{EC}}$ to determine the relative importance of
234 building operation modes and choice of baselines on the discrepancy between $Q_{F,B}$ and Q_{EC} .

235 3 Results and discussion

236 Building energy balance fluxes vary through each day and season (Fig. 1) associated with when a building is
237 occupied and people's activities inside the building. First, we consider one case in detail – an occupied building
238 with both natural ventilation and HVAC (*ov3*, Table 1) relative to an unoccupied sealed building (*us*, Table 1) –
239 their difference (*ov3-us*) allows us to obtain the fluxes needed (Sect. 1).

240 As noted (Sect. 1), the shortwave and incoming longwave radiative fluxes for all cases (Table 1) are
241 assumed identical, but all other terms of the building energy balance differ. Hence, the change in outgoing
242 longwave radiation ($\Delta L_{\uparrow,o-uo}$, Fig. 1c) is equivalent to the net all-wave radiation difference (Q_{o-uo}^* , Fig. 1a-b)
243 for the occupied and unoccupied buildings. The positive sensible heat flux difference (Eq. (10), ΔH_{o-uo} , Fig. 1c)
244 and $\Delta L_{\uparrow,o-uo}$ indicate the building is warmed up by internal heat gains ($Q_{Internal,o}$) with higher exterior surface
245 temperatures. Their small magnitudes and flat patterns indicate small relative importance compared to the heat
246 exchange from ventilation differences (Eq. (8), ΔBAE_{o-uo} , Fig. 1c). The latter, not only contributes the largest
247 fraction of anthropogenic heat flux ($Q_{F,B}$, Fig. 1c), but also has a diurnal pattern consistent with $Q_{F,B}$, especially
248 during spring and autumn (Fig. 1c, i). Rarely, heat ($Q_{Waste,o}$, Fig. 1i) is emitted by the air conditioner in the
249 mid-afternoon (shading) at this time of year, but more importantly in summer (Fig. 1f) when cooling demand
250 increases.

251 $Q_{F,B}$ (Eq. (14), Fig. 1c) has four components of emitted heat, whereas energy consumption (Q_{EC} , Fig. 1c)
252 only has (in this case, constant) internal heat gains ($Q_{Internal,o} = 11.8 \text{ W m}^{-2}$, Fig. 1b, Table 1) and energy use
253 from HVAC system (Q_{HVAC} , Fig. 1b). Their difference is the storage heat flux difference (Eq. (15) ΔS_{o-uo} in Fig.
254 1c). If ΔS_{o-uo} is positive, the building acts as a heat sink and stores the extra heat generated by human activities,
255 or stored heat is released when ΔS_{o-uo} is negative. Hence, we can identify the impacts of seasonal-varying
256 human activities and building operations on the diurnal variability in ΔS_{o-uo} , Q_{EC} and $Q_{F,B}$.



257

258 Figure 1: Seasonal diurnal median (line) and inter-quantile range (IQR, shading) building heat fluxes for (a, d, g, j)
 259 unoccupied sealed (us), (b, e, h, k) occupied ventilated (ov3) building and their (c, f, i, l) difference (ov3-us) for (a-c) spring,
 260 (d-f) summer, (g-i) autumn and (j-l) winter. $Q_{F,B}$ is estimated by either heat transfer difference (solid line components):
 261 $Q_{F,B} = \Delta L_{1,o-uo} + \Delta H_{o-uo} + \Delta BAE_{o-uo} + Q_{Waste,o}$ in Eq. (14) or energy consumption and storage flux difference: $Q_{F,B} =$
 262 $Q_{EC} - \Delta S_{o-uo}$ (dash line components) in Eq. (15)

263 3.1 Impact of human activities on seasonal and diurnal variations of the fluxes

264 For the same *ov3-us* case (Table 1, Fig. 1), we consider the diurnal and seasonal variability of the fluxes. In
265 spring and autumn (Fig. 1a-c, g-i), natural ventilation is the dominant factor contributing to diurnal variation in
266 ΔS_{o-u_0} and $Q_{F,B}$, while Q_{EC} has minimal variability. Q_{EC} is slightly larger than $Q_{Internal,o}$ because of some
267 short periods of HVAC use in the mid-afternoon (IQR shading in Fig. 1i). There is a clear diurnal cycle of $Q_{F,B}$
268 (Fig. 1c) with the median varying between 8 W m^{-2} (07:00) and 15 W m^{-2} (15:00) relative to the constant
269 internal heat gain (11.8 W m^{-2}). The difference between $Q_{F,B}$ and Q_{EC} (ΔS_{o-u_0}) is largely impacted by natural
270 ventilation. During the night and early morning with closed window, only part of the consumed energy is
271 transferred externally to the atmosphere. The rest of the heat is stored in the building fabric (positive ΔS_{o-u_0}),
272 hence $Q_{F,B}$ is lower than Q_{EC} . However, when overheating may occur during the middle of the day, occupants
273 keep window opened (air conditioner is less frequently used) to cool the building down, with stored heat
274 released (negative ΔS_{o-u_0}). This is consistent with the diurnal variability of ΔBAE_{o-u_0} which has a minimum at
275 night (window closed) and maximum in the mid-noon (window open).

276 In summer, the daytime natural ventilation is replaced by air conditioning as natural ventilation alone could
277 not maintain thermal comfort indoors. Natural ventilation and waste heat from the air conditioner ($Q_{Waste,o}$)
278 contribute to one peak $Q_{F,B}$ at nighttime and daytime, respectively (Fig. 1f). $Q_{F,B}$ is higher than Q_{EC} around
279 these two peak periods (05:00-07:00 and 13:00-21:00). The peak $Q_{F,B}$ at night reaches 14 W m^{-2} (median) at
280 05:00, which is mainly attributed to natural ventilation when outdoor air temperature is cooler than indoors.
281 Conversely, in the afternoon when outdoor temperature is warmer, occupants ‘choose’ mechanical cooling for
282 achieving thermal comfort. The peak $Q_{F,B}$ is 22 W m^{-2} at 16:00, approximately 22% higher than Q_{EC} . It
283 indicates that using Q_{EC} for the anthropogenic heat flux from buildings (e.g. Heiple and Sailor, 2008) may
284 underestimate the effect of $Q_{F,B}$ on urban atmospheric processes especially during the late afternoon/early
285 evening. In addition, $Q_{F,B}$ is always smaller than $Q_{Waste,o}$ because of the negative $\Delta L_{\uparrow,o-u_0}$ and ΔH_{o-u_0} causing
286 a cooler exterior surface. This suggests using $Q_{Waste,o}$ as $Q_{F,B}$ (e.g. Chow et al., 2014) may overestimate $Q_{F,B}$ in
287 summer.

288 However, in winter, mechanical heating and thermal mass effect shape the temporal pattern of $Q_{F,B}$ (Fig.
289 1i). The cool outdoor air temperature before sunrise results in a substantial heating supply and peak Q_{EC} (16.43
290 W m^{-2} for median line) at 08:00. This heat is stored in building fabric (positive ΔS_{o-u_0}) and have a relatively

291 stable release through convection and longwave radiation. Therefore the diurnal profile $Q_{F,B}$ is rather flatter and
292 ΔS_{o-u0} has a highly consistent temporal pattern to Q_{EC} .

293 Overall, this analysis recognizes the crucial role of ΔS_{o-u0} in distinguishing $Q_{F,B}$ from Q_{EC} , which is highly
294 dependent on HVAC operation and natural ventilation (i.e., human activity of window opening). These two
295 factors can rapidly increase or decrease $Q_{F,B}$ while convection and longwave radiation cannot. Whereas in
296 winter, the larger IQR (shading) of $Q_{F,B}$ than Q_{EC} indicates more day-to-day variation in $Q_{F,B}$ diurnal profile
297 than Q_{EC} . Estimates of $Q_{F,B}$ using satellite remote sensing found heat storage plays an important role in
298 moderating energy use within buildings (Yu et al., 2021). As the storage heat flux change modifies the diurnal
299 sensible heat flux pattern it modifies the surface temperature increment ($Q_{F,B}$ in remote sensing approach) and
300 hence the apparent energy consumption.

301 The diurnal profiles of ΔS_{o-u0} are not identical between seasons as people use different actions to achieve
302 thermal comfort in different weather conditions. This suggests the $Q_{F,B}$ and Q_{EC} differences may vary between
303 climates and with cultural practices. In inventory methods the diurnal profiles may be limited (e.g. LUCY (Allen
304 et al., 2011), weekday/weekend by country) and ignore seasonal variations. However, ΔS_{o-u0} behaviour types
305 classes may benefit from distinguishing diurnal variation for different climates.

306 **3.2 Impact of different building operation modes on seasonal and diurnal variations**

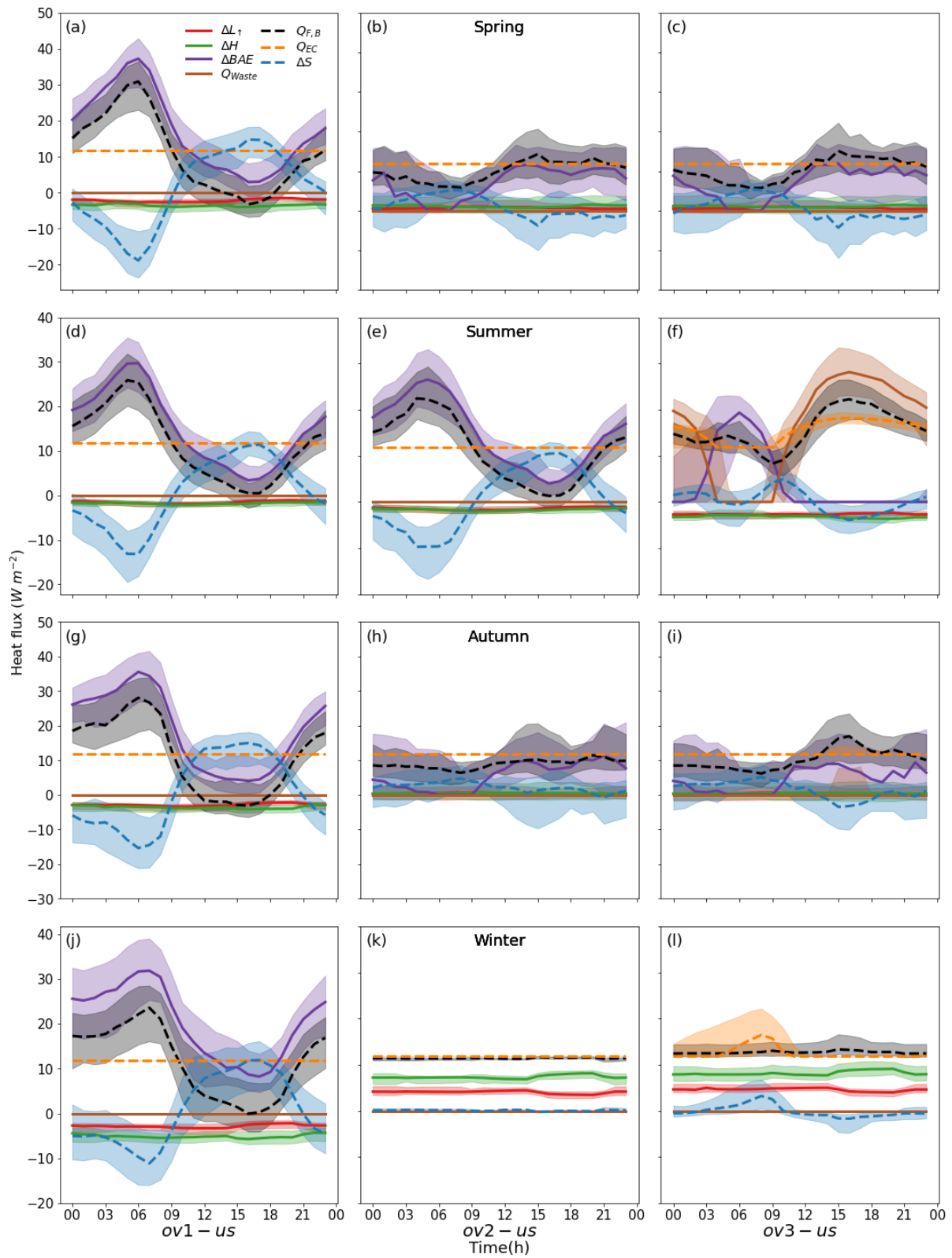
307 Fig. 2 illustrates the impact of different building operation modes (Table 1: *ov1*, *ov2*, *ov3*; cf. *us*) on the $Q_{F,B}$
308 diurnal profiles. It suggests the different ventilation strategies and HVAC systems do change $Q_{F,B}$ in both
309 temporal pattern and magnitude, but their impacts vary among seasons.

310 In spring and autumn, different natural ventilation control strategies completely modify the $Q_{F,B}$ diurnal
311 profile, whereas HVAC system only increases the peak $Q_{F,B}$ slightly in autumn (Fig. 2i). The distinctly different
312 (opposite) trend in diurnal $Q_{F,B}$ pattern for *ov1* cf. *ov2* or *ov3* (Fig. 2a-c, g-i) is largely explained by the diurnal
313 change of ΔBAE_{o-u0} in the three cases. In *ov1* (window open, no control) the minimum outdoor air temperature
314 before sunrise creates the maximum indoor and outdoor air temperature difference, therefore the highest
315 ΔBAE_{o-u0} and peak $Q_{F,B}$ at 06:00 (30 W m⁻² for the median in Fig. 2a). Whereas *ov2* and *ov3* have the window
316 closed at night and early morning to avoid overcooling, therefore the minimum $Q_{F,B}$ in the early morning
317 (07:00). As outdoor air temperature increases through the day, $Q_{F,B}$ follows the reduced ΔBAE_{o-u0} in *ov1*,
318 whereas natural ventilation is active in *ov2* and *ov3*, leading to an increase in ΔBAE_{o-u0} and $Q_{F,B}$. Unlike *ov2*,
319 *ov3* has a clear peak (16 W m⁻² median, Fig. 2i) at 15:00, because when natural ventilation alone cannot satisfy

320 thermal comfort and *ov3* air conditioning is activated. But their overall patterns (IQR) are very consistent,
 321 indicating afternoon use of air conditioning could increase $Q_{F,B}$ magnitude but have a limited impact on other
 322 parts of the diurnal pattern. Surprisingly, negative $Q_{F,B}$ occurs around 17:00 in spring (Fig. 2a), suggesting the
 323 occupied building has less heat emissions than unoccupied building. Because the natural ventilation at night and
 324 morning cools down the building and reduced fabric exterior surface temperature leads to a large reduction in
 325 longwave radiation and convection ($\Delta L_{\uparrow,o-u0}$ and ΔH_{o-u0}) than increase in heat emission through natural
 326 ventilation (ΔBAE_{o-u0}) in afternoon. And the reduced overall emissions are converted into increase in storage
 327 heat flux (ΔS_{o-u0}). Negative $Q_{F,B}$ also occurs when unoccupied building is always ventilated (*uv*) and occupied
 328 building is ventilated with control (*ov2* and *ov3*) in spring (e.g. Fig. B6b-c). The window is closed to avoid
 329 excessive cooling at night in *ov2*. With ΔBAE_{o-u0} negative in this case, its magnitude is much larger than
 330 increase in longwave radiation and convection ($\Delta L_{\uparrow,o-u0}$ and ΔH_{o-u0}). The minimum $Q_{F,B}$ frequently
 331 corresponds to the peak ΔS_{o-u0} .

332 In summer, *ov2* window is open most of the time (as in *ov1*) for thermal comfort, therefore the $Q_{F,B}$ has no
 333 apparent difference to *ov1*. However for *ov3*, as air conditioning runs from morning to late night and there is a
 334 very different diurnal profile (cf. *ov2* and *ov1*). Air conditioner use contributes to a much larger $Q_{F,B}$ (cf. *ov2*)
 335 from 12:00 to 21:00. Not only is extra energy consumed, but it also removes heat from building to the
 336 atmosphere in this period. In contrast, using natural ventilation as a cooling strategy (*ov1* and *ov2*) contributes to
 337 a high $Q_{F,B}$ at night and early morning but very low even negative extra heat emission in afternoon. This implies
 338 natural ventilation as passive cooling strategy not only could improve the thermal conditions indoors, but also
 339 could contribute to the improvement of outdoor climate by modifying the diurnal pattern of anthropogenic heat
 340 emissions (Duan et al., 2019).

341 Consistent with results in the other seasons, different ventilation control strategies in winter cause a large
 342 change in $Q_{F,B}$ profile between *ov1* and *ov2*. However, the temporal pattern of $Q_{F,B}$ (IQR) in *ov2* is quite similar
 343 to *ov3* because the supplied heat from mechanical heating system does not immediately enhance $Q_{F,B}$ with
 344 closed window. *ov2* is the only scenario that has similar $Q_{F,B}$ and Q_{EC} through the whole day. Comparison using
 345 an unoccupied ventilated (*uv*) baseline (Fig. B.6) (cf. *Us* Fig. 2) show that although $Q_{F,B}$ profiles differ, the
 346 impacts of different building operation modes are consistent when the same occupied buildings used. The
 347 impact of baselines with different air exchange on $Q_{F,B}$ are analysed in Sect. 3.3.



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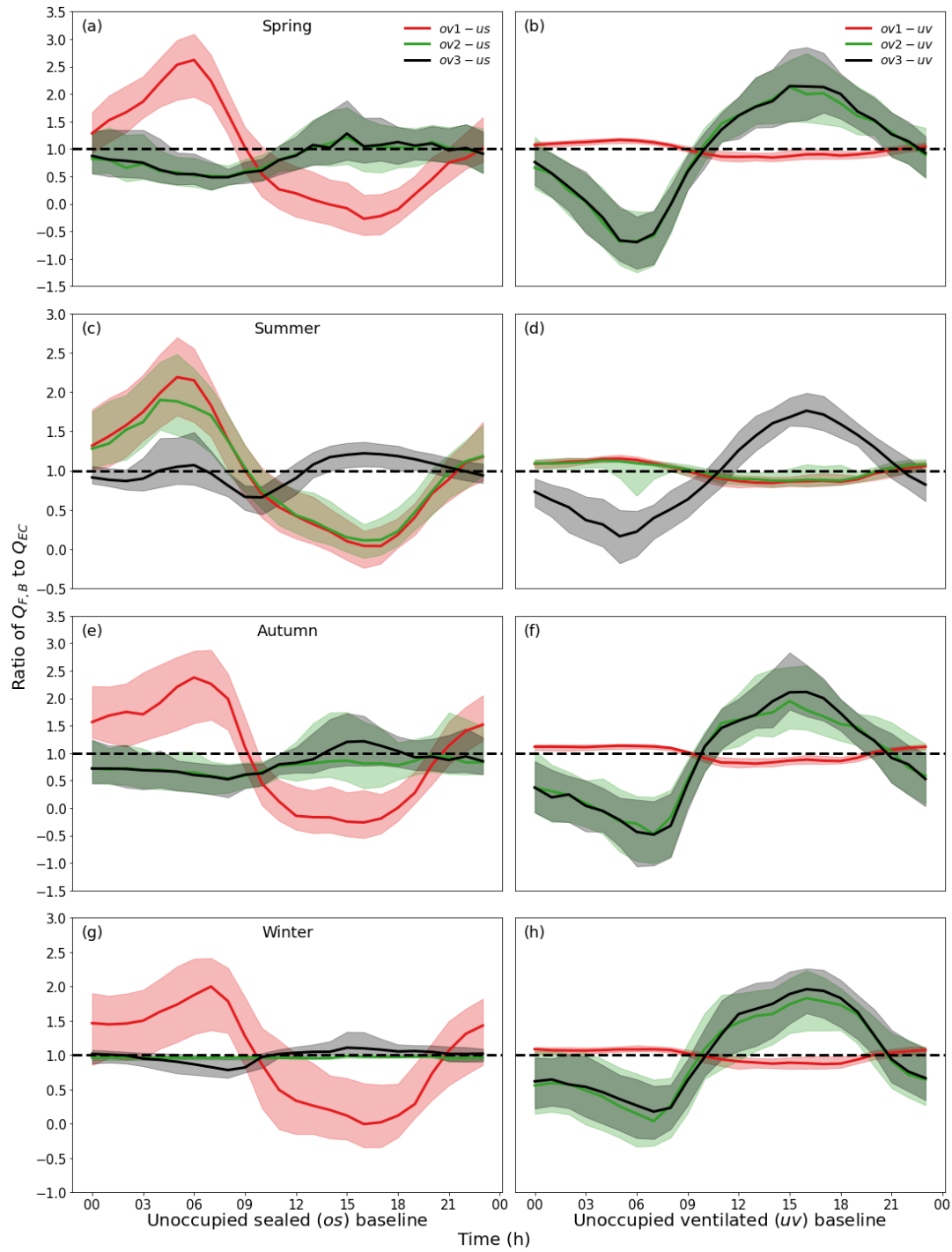
Figure 2 : As Figure 1c, f, i, j, but comparing three different building operation types (a, d, g, j) *ov1*: window is always open without control, no HVAC; (b, e, h, k) *ov2*: controlled natural ventilation for indoor thermal comfort, no HVAC; (c, f, i, l) *ov3*: mixed mode ventilation

352 3.3 Impact of unoccupied baseline chosen

353 Here two unoccupied baselines (*us* – unoccupied sealed building, *uv* – unoccupied ventilated building with
354 uncontrolled open window) are used to assess the impact. A ratio between $Q_{F,B}$ to Q_{EC} is used (Fig. 3) to
355 normalize the impact of baselines on their difference with different building operation modes. The largest
356 difference in R occurs on 23 December at 11:00, with values of 5.13 (*ov3-uv*) and -2.72 (*ov1-us*), reflecting the
357 considerable difference between $Q_{F,B}$ to Q_{EC} .

358 Two diurnal patterns of the R ratio are distinguished. When the window is always open (*ov1* in all seasons,
359 *ov2* in summer), $R > 1$ ($Q_{F,B} > Q_{EC}$) at night/early morning (22:00-08:00), reaching its maximum around
360 05:00-07:00 (near sunrise in all seasons). For the remaining periods, which are relatively warm, $R < 1$. Whereas,
361 when window opening/closing is controlled and HVAC is used for thermal comfort an almost inverse temporal
362 pattern of R occurs, with $R > 1$ during afternoon when either window is open or the air conditioner is activated.
363 The peak R occurs at 15:00 when both outdoor temperature and solar radiation are high.

364 When different unoccupied baselines are used, the temporal patterns of R are similar for all cases, but their
365 magnitudes differ significantly. R is close to 1 when window states between unoccupied and occupied buildings
366 are similar (e.g. *ov1-uv* in all seasons, *ov2-uv* in summer). Hence, greater difference occurs in heat transfer from
367 ventilation or mechanical heating/cooling between occupied and unoccupied building (i.e., larger R). Thus, the
368 baseline chosen impacts the results and require appropriate consideration for incorporating $Q_{F,B}$ into
369 atmospheric modelling.

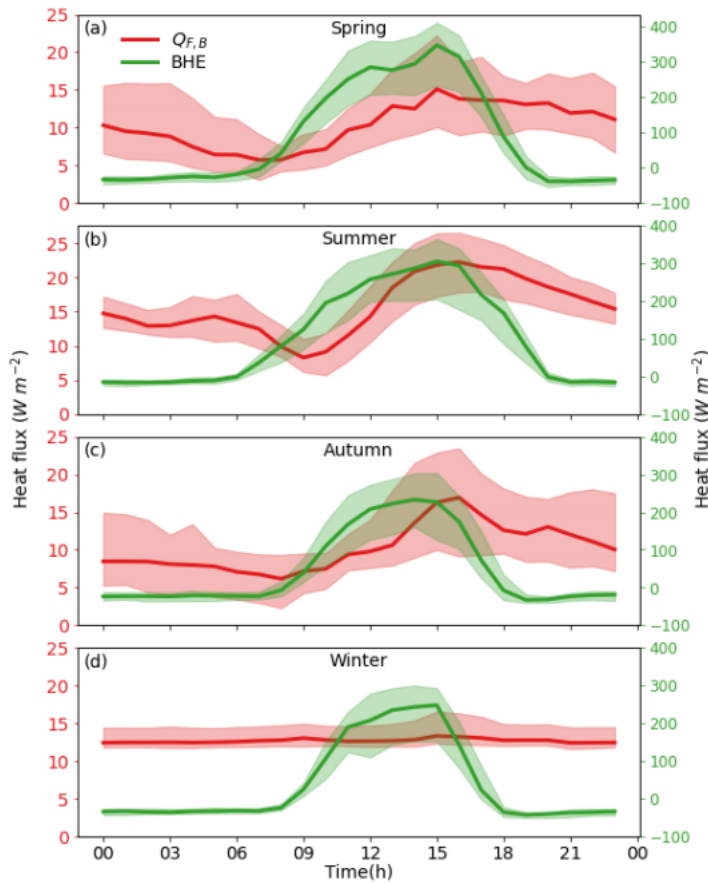


370

371 Figure 3: $Q_{F,B}$ to Q_{EC} ratio I median (line) and IQR (shading) for (a-b) spring, (c-d) summer, (e-f) autumn and (g-h) winter,
 372 using two unoccupied baselines: (a, c, e, g) sealed (us), and (b, d, f, h) ventilation (uv); each with three occupancy types
 373 (colour): $ov1$: Only internal heat gains are applied and window is fully open; $ov2$: Internal heat gains and natural ventilation
 374 control are applied. $ov3$: Internal heat gains, natural ventilation control and HVAC system are applied. Ratio $R=1$ (Black
 375 dotted line)

376 **3.4 Comparison between $Q_{F,B}$ and building heat emission (BHM)**

377 Comparison of building heat emissions (BHE), determined using the Hong et al. (2021) approach, to $Q_{F,B}$
378 (this study) for one case (*ov3-us*) shows that the former is much larger than $Q_{F,B}$ during the day but smaller at
379 night and have different diurnal patterns (Fig. 4). Convection from the exterior envelope (Q_H , Figure 1b, e, h, k)
380 is the main contributor to BHE, therefore influences the BHE diurnal profile in each season. During the day,
381 solar radiation is large and a major control whereas $Q_{F,B}$ is relatively small and consistent but modified by
382 building-human interactions (e.g., opening windows, activation of mechanical heating and cooling systems). In
383 this scenario shown, natural ventilation and mechanical cooling dominate $Q_{F,B}$ in summer and shoulder season;
384 while in winter in their absence, convection and longwave radiation are more important.



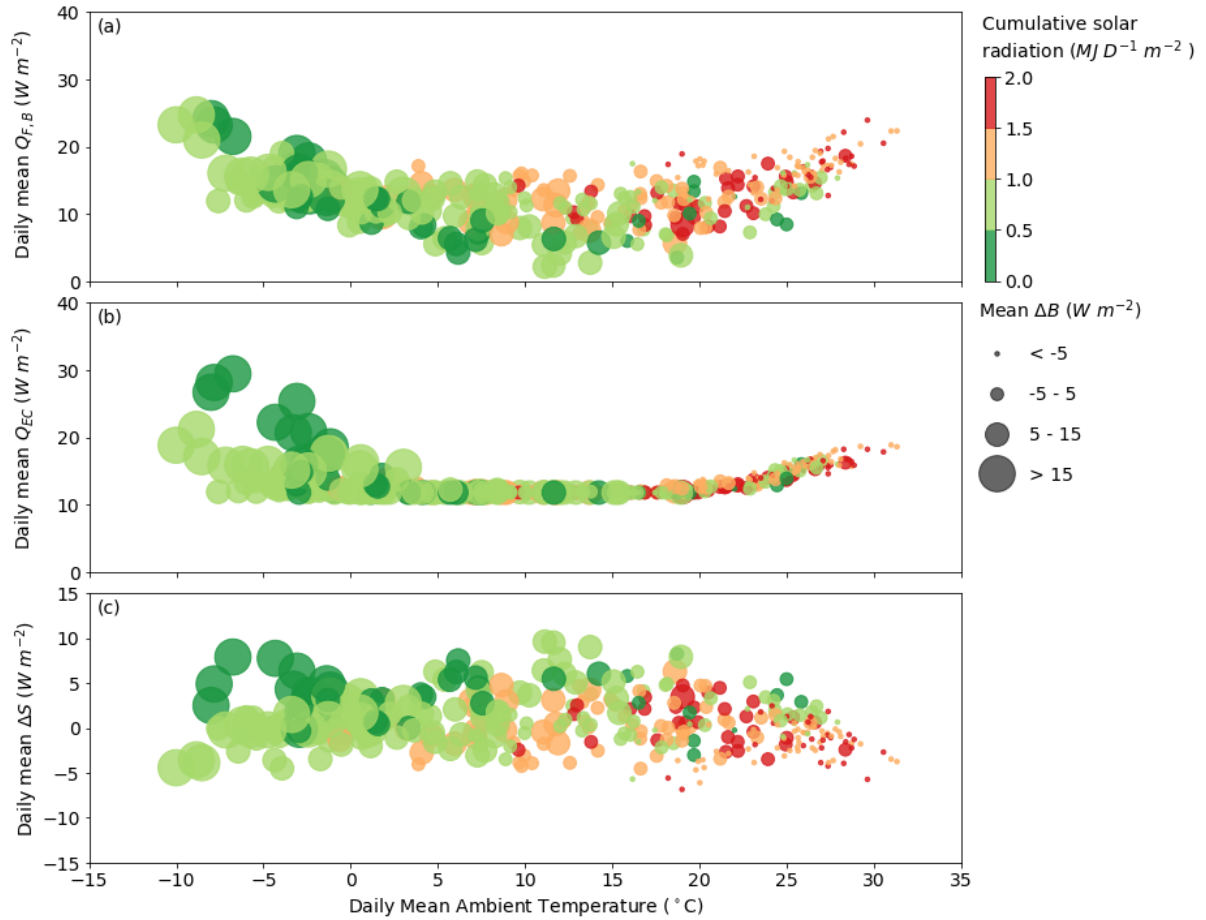
385
386 Figure 4: Comparison of seasonal diurnal $Q_{F,B}$ (*ov3-us*) and building heat emission (BHE, *ov3* in Table1) for (a) spring, (b)
387 summer, (c) autumn and (d) winter.

388 **3.5 Daily variation of fluxes in relation to meteorological conditions**

389 Ambient air temperature is one of the most crucial factors controlling building energy consumption (Sailor and
390 Vasireddy, 2006). Hence, it is often used to determine daily variability of Q_{EC} (e.g. Lindberg et al., 2013) and
391 the resulting monthly variations (e.g. Allen et al., 2011). By accounting for ΔS_{o-u_0} in this study, the response of

392 $Q_{F,B}$ to ambient air temperature may differ to previous studies. To examine this we use the *ov3-us* case to
393 consider the relations of daily mean (unless indicated) variables of air temperature (mean) , solar radiation (daily
394 total) and simulated available energy to the building from human activities (ΔB) with anthropogenic heat flux
395 ($Q_{F,B}$ in Fig. 5a), energy consumption (Q_{EC} in Fig. 5b) and their difference (ΔS_{o-u0} in Fig. 5c). The overall trends
396 between $Q_{F,B}$ and Q_{EC} to ambient air temperature are consistent, with $Q_{F,B}$ and Q_{EC} smallest when temperatures
397 are between 10-15°C. This coincides with the Nicol and Humphreys'(2002) monthly balance-point temperature
398 of 12°C, which has been regarded as the equivalent ambient air temperature with the minimum energy use
399 within the building (e.g. Allen et al., 2011, Koralegedara et al., 2016). As the temperature increases (decreases),
400 Q_{EC} increases proportionally with temperature due to mechanical cooling (heating). However, in contrast to
401 Q_{EC} , $Q_{F,B}$ has a much larger variability at the same temperature caused by a large range of ΔS_{o-u0} (-7.7 to 9.0 W
402 m⁻²), which is highly dependent on human activities on diurnal scale (Sect. 3.1)

403 To understand the large daily variability of ΔS_{o-u0} , we use ΔB (net available energy from human activities
404 in buildings in Eq. 9) to indicate the effect of human activities (heat addition or removal) in one day. Higher ΔB
405 (larger circles) are associated with higher ΔS_{o-u0} at the same ambient air temperature, especially in winter (Fig.
406 5c). This is not unexpected as buildings will absorb more heat when extra internal energy is added into the
407 building. Inversely, negative ΔB (small circles) contributes to much more heat release from heat storage (lower
408 ΔS_{o-u0} through either natural ventilation or mechanical cooling. The sign and magnitude of ΔB are linked to
409 daily cumulative solar radiation. At the same ambient air temperature, more solar radiation enhances the need
410 for larger heat removal or less heat addition to the building for thermal comfort, therefore leading to a smaller
411 ΔB and lower ΔS_{o-u0} . Consequently, we can conclude that both ambient air temperature and cumulative solar
412 radiation are important meteorological factors to determining ΔS_{o-u0} and $Q_{F,B}$.



413

414 Figure 5: Daily results for the *ov3-us* case stratified by daily cumulative solar radiation (colour) and daily mean available
 415 energy to the building (size) (Eq. (9) associated with human activities, with mean external air (ambient) temperature and (a)
 416 mean anthropogenic heat flux, (b) energy consumption and (c) difference in storage heat flux.

417 4 Conclusions

418 Anthropogenic heat flux from buildings ($Q_{F,B}$) is defined as the additional heat released from building into
 419 atmosphere due to human activities. It is qualitatively and quantitatively different to building energy
 420 consumption (Q_{EC}) in temporal pattern and magnitude as result of thermal inertia of building (Iamarino et al.,
 421 2012). However, as there is no standard to quantify ‘real’ $Q_{F,B}$ most studies use Q_{EC} as a proxy via inventory
 422 and building energy modelling approaches. This paper proposes a new method to quantify a more appropriate
 423 $Q_{F,B}$ by utilising the difference in heat fluxes between an occupied and unoccupied building (i.e. the built
 424 structure with absolutely no energy use and human metabolism). We show the difference between Q_{EC} and $Q_{F,B}$
 425 is attributable to a change in the storage heat flux induced by human activities (ΔS_{O-UO}). $Q_{F,B}$ has four
 426 components based on its dissipation pathways, including outgoing longwave radiation, turbulent sensible heat
 427 flux (convection), heat release due to air exchange and waste heat from HVAC systems. We use one simplified

428 case study in Beijing to demonstrate the analysis using building energy simulations to quantify the temporal
429 difference between Q_{EC} and $Q_{F,B}$ and understand the relative importance of building operations for thermal
430 comfort and meteorological condition on $Q_{F,B}$. The key conclusions are:

- 431 • Hourly ratios between $Q_{F,B}$ and Q_{EC} can differ between -2.72 and 5.13 because of differences in
432 occupancy use of the building (within a year, in Beijing's climate). Individual ratios frequently exceed
433 3 between 14:00 and 16:00 when controlled natural ventilation or mechanical cooling is activated in
434 shoulder season). Thus, the definitions differences are large.
- 435 • Natural ventilation (ΔBAE_{o-u0}) or HVAC operation ($Q_{Waste,o}$ for cooling and Q_{HVAC} for heating) are
436 two predominant contributors to the storage heat flux. Hence, different building operations to control
437 thermal comfort determine the diurnal profile of $Q_{F,B}$ by affecting not only Q_{EC} but also ΔS_{o-u0} .
- 438 • The day-to-day variation of $Q_{F,B}$ diurnal profile is broader than that of Q_{EC} .
- 439 • Diurnal profile of ΔS_{o-u0} varies with season as occupants modify their behaviours and the interaction
440 with buildings to achieve thermal comfort (e.g. cooling in summer and heating in winter), indicating
441 differences between $Q_{F,B}$ and Q_{EC} will vary with both climate and cultural norms.
- 442 • $Q_{F,B}$ is sensitive to the unoccupied baseline chosen (here two are analysed unoccupied sealed vs
443 unoccupied ventilated). An 'unoccupied baseline' needs to be integrated into urban climate modelling
444 in the future.
- 445 • Daily mean temperature only accounts for the day-to-day variability in Q_{EC} rather than ΔS_{o-u0} . Both
446 ambient air temperature and cumulative solar radiation are important meteorological factors to
447 determine ΔS_{o-u0} and $Q_{F,B}$.

448 Our new approach should be used to provide data for future parameterisations of both anthropogenic heat
449 flux from buildings and storage heat fluxes for urban weather and climate modelling. We conclude that storage
450 heat fluxes in cities is also being modified by occupant behaviour, particularly by natural ventilation and
451 mechanical cooling. It is expected that the diurnal variation of ΔS_{o-u0} will vary with operation schedules for
452 different building uses (e.g. residential vs. commercial buildings). Given the release of stored heat is critical
453 influence on the nocturnal canopy layer urban heat island (CL-UHI), the impact of different HVAC operations
454 on nocturnal UHI should be explored further. This is an important factor to determine diurnal pattern of $Q_{F,B}$ in
455 the shoulder season and can be expressed more accurately. However, in different climates and with different
456 social cultural practices the periods most influenced will change. Further studies are being conducted to explore
457 the impacts of these, while also addressing feedbacks at the neighbourhood scale.

458 For developers of urban canopy parameterisations (UCP) there are several considerations because of
459 computational efficiencies essential for undertaking weather and climate modelling: (1) human activities within
460 building are modifying both the storage heat flux and the anthropogenic heat flux; (2) assuming within an UCP
461 that a ‘simple’ building energy model (BEM) (cf. a full building energy simulation scheme such as EnergyPlus)
462 will require some human activities to be simplified, such as using fixed ventilation rate, instead of dynamic
463 natural ventilation depending on both outdoor weather condition and thermal comfort requirements; and (3) with
464 a multi-layer UCP the appropriate levels for the impact of these energy exchanges can be accounted for. Our
465 current research is extending this analysis to consider moisture; and exploring the role of building materials,
466 construction, other aspects of building design and external meteorology. The outcome of this work will also
467 have implications for UCP development, as can help identify what can be simplified and what are critical
468 controls in different climates and urban settings.

469 **Data availability**

470 All data are deposited at <https://doi.org/10.5281/zenodo.5903303> (Liu et al., 2022)

471 **Author contributions**

472 Conceptualisation: SG and ZL, Methods and Analysis: YL SG and ZL, First draft and visualization: YL,

473 Writing and review for submission: YL, SG and ZL Funding: SG and ZL.

474 **Competing interests**

475 The author declare that they have no conflict of interest.

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483 **Nomenclature**

| | |
|----------------------------------|---|
| A_{eff} | Effective area of windows opening (m^2) |
| ΔB_{o-uo} | Available energy to the building from human activities ($W\ m^{-2}$) |
| ΔBAE_{o-uo} | Difference of heat transfer by air exchange between building and atmosphere between occupied (o) and unoccupied (uo) building ($W\ m^{-2}$) |
| BHE | Building heat emission to ambient air ($W\ m^{-2}$) |
| ΔH_{o-uo} | Difference in Q_H between occupied (o) and unoccupied (uo) building ($W\ m^{-2}$) |
| $F_{[sky \rightarrow boi]}$ | View factor from sky to building of interest |
| $F_{[other\ b \rightarrow boi]}$ | View factor from other buildings to building of interest |
| $F_{[boi \rightarrow sky]}$ | View factor from building of interest to sky |
| $F_{[boi \rightarrow other\ b]}$ | View factor from building of interest to other buildings |
| C_d | Discharge coefficient |
| H | Height of windows opening (m) |
| K_{\uparrow} | Outgoing shortwave radiative flux ($W\ m^{-2}$) |
| K_{\downarrow} | Incoming shortwave radiative flux ($W\ m^{-2}$) |
| L_{\uparrow} | Outgoing longwave radiative flux ($W\ m^{-2}$) |
| L_{\downarrow} | Incoming longwave radiative flux ($W\ m^{-2}$) |
| $\Delta L_{\uparrow, o-uo}$ | Difference in L_{\uparrow} between occupied (o) and unoccupied (uo) building ($W\ m^{-2}$) |
| ΔQ_S | Net storage heat flux for the building volume ($W\ m^{-2}$) |
| Q^* | Net all-wave radiative flux ($W\ m^{-2}$) |
| Q_{AC} | Sensible cooling load from air conditioning ($W\ m^{-2}$) |
| Q_{BAE} | Heat transfer by air exchange between building and atmosphere ($W\ m^{-2}$) |
| $Q_{F, B}$ | Anthropogenic heat flux from building sector ($W\ m^{-2}$) |
| $Q_{F, M}$ | Anthropogenic heat flux from metabolic activities ($W\ m^{-2}$) |
| $Q_{F, T}$ | Anthropogenic heat flux from transport ($W\ m^{-2}$) |
| Q_H | Turbulent sensible heat flux ($W\ m^{-2}$) |
| Q_{HS} | Sensible heating load ($W\ m^{-2}$) |
| Q_{HVAC} | Energy consumption by heating ventilation and air conditioning (HVAC) system ($W\ m^{-2}$) |
| $Q_{Internal}$ | Internal heat gain within the building (human metabolism, lighting and appliance) ($W\ m^{-2}$) |
| Q_{Waste} | Waste heat released to outdoor by HVAC system ($W\ m^{-2}$) |
| R | Ratio of anthropogenic heat flux from building ($Q_{F, B}$) to energy consumption (Q_{EC}) |
| ΔS_{o-uo} | Different in storage heat flux between occupied (o) and unoccupied (uo) building ($W\ m^{-2}$) |
| T_{ave} | Average indoor and outdoor air temperature ($^{\circ}C$) |
| ΔT | Indoor and outdoor air temperature difference ($^{\circ}C$) |
| U_W | Reference wind speed at height of upstream airflow ($m\ s^{-1}$) |
| V_{Stack} | Buoyance driven ventilation rate ($m^3\ s^{-1}$) |
| V_T | Total ventilation rate by combined wind and bouyance effect |
| V_W | Wind driven ventilation rate ($m^3\ s^{-1}$) |

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